

Ventiduct® nozzle ducts



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Ventiduct® nozzle ducts

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Nozzle ducts

Air distribution system

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VSR



407

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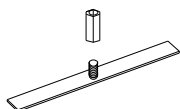
4

Nozzle ducts - accessories

Air distribution system

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Accessories

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Ventiduct® nozzle ducts



Office building, Copenhagen

Ventiduct® nozzle ducts

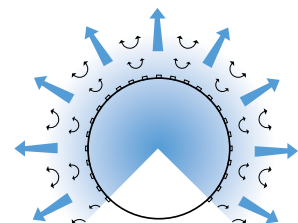
Air distribution by way of nozzle ducts can be used to great effect in all places that need chilled air. This means from industrial places to comfort-projects. Consequently, it is a great alternative to the more traditional supply-air diffusers. This is especially so, when a visible installation is desired as functional decor. Those rooms where Ventiduct nozzle ducts are not used, an alternative is possible by using unperforated spiral ducts. Ventiduct is a strong alternative to textile-ducts.

Principle of ventilation

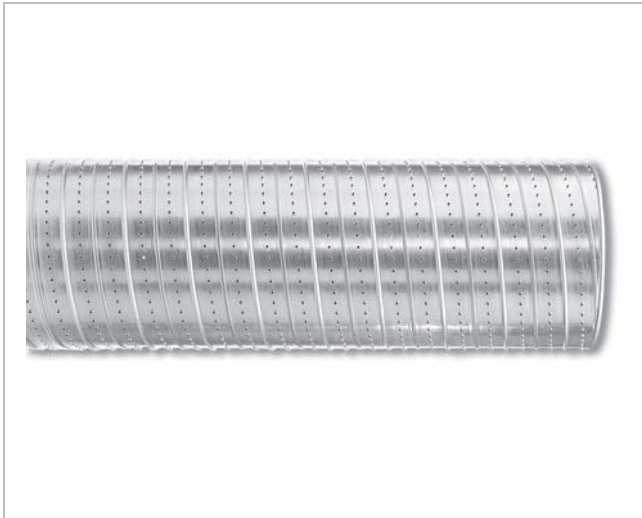
The principle of ventilation is a form of active thermal displacement, suited for ventilation and cooling purposes. The air pattern in active thermal displacement consists of both up-ward directed and down-ward directed airflows. The name means that the airflow in the occupied zone is caused by the supply air diffuser and the heating sources.

Function

Ventiduct nozzle ducts give a great cooling effect, since it can be supplied with a cooling temperature of up to approx. 10 K. The supplied air is supplied with great velocity through the many small nozzles in the duct-walls. This creates a good induction. Ventiduct has a large area of dynamics, which makes it possible to regulate the air-volume / air flow from 30 - 100%.



Ventiduct® nozzle duct - cross section



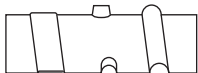
Description

Ventiduct is an air distribution system consisting of spiral-seamed circular ducts that is equipped with a large number of small nozzles inserted into the duct wall. They are supplied in five sizes from $\varnothing 200$ mm to $\varnothing 500$ mm and with various nozzle patterns, which should be chosen according to the task in hand.

Maximum standard length is 3,000 mm. The ducts have a raised protective cover to prevent the nozzles becoming deformed during transport. Ventiduct ducts can be supplied in hot-galvanised or powder-coated versions. The system should be primarily used for the supply of cooled air.

- Large cooling effect
- Large dynamic range
- Large induction rate
- Short throw
- Discrete diffuser design
- Easy to install

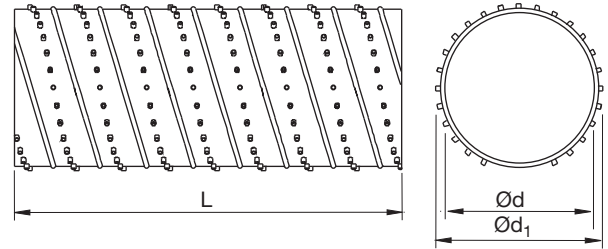
Cross-section of nozzle duct



Order code

Product	VSR	aaa	bbb	cccc	d/e
Type					
$\varnothing d$					
Nozzle pattern					
Length/no. of parts					
Finish					
0 Galvanised					
1 Powder coated					

Dimensions

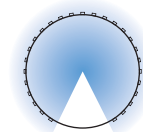


$\varnothing d$ mm	$\varnothing d_1$ mm	L mm	Weight kg
200	212	3000	4,5
250	262	3000	5,4
315	327	3000	6,9
400	412	3000	8,6
500	512	3000	10,9

Nozzle pattern

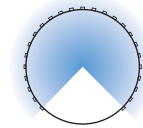
Code

300°



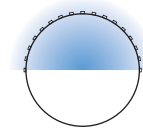
300

270°



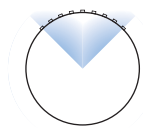
270

180°



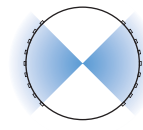
180

90°



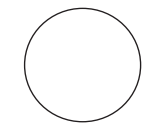
090

2 x 90°



290

Blind piece
without nozzles :
spiral-seamed
long-seamed



000
001

The blind piece is a specially made spiral-seamed duct that resembles ventiduct in design, as it has no actual nozzles.

Available in the same length as ordinary nozzle ducts.

Alternatively long-seamed pipes can be used, which creates an attractive contrasting effect.

Dispersal patterns

With Ventiduct nozzle ducts, various flow conditions can be achieved in the room. The downward supply of air always creates the greatest air velocities in the occupied zone and is therefore used mostly in industrial ventilation. The choice between air being supplied horizontally or upwards depends on the required form of flow.

Upward supply air

When cooled air is supplied upwards, the cool air mixes with the warmer room air close to the duct nozzles. The supplied air typically covers a vertical area of 2-4 metres below the ducts. At greater distances between the ducts, the supplied air flows behind in a displacement flow further out in the room.

Depending on the required volume flow, a nozzle pattern of between 90° and 300° is used.

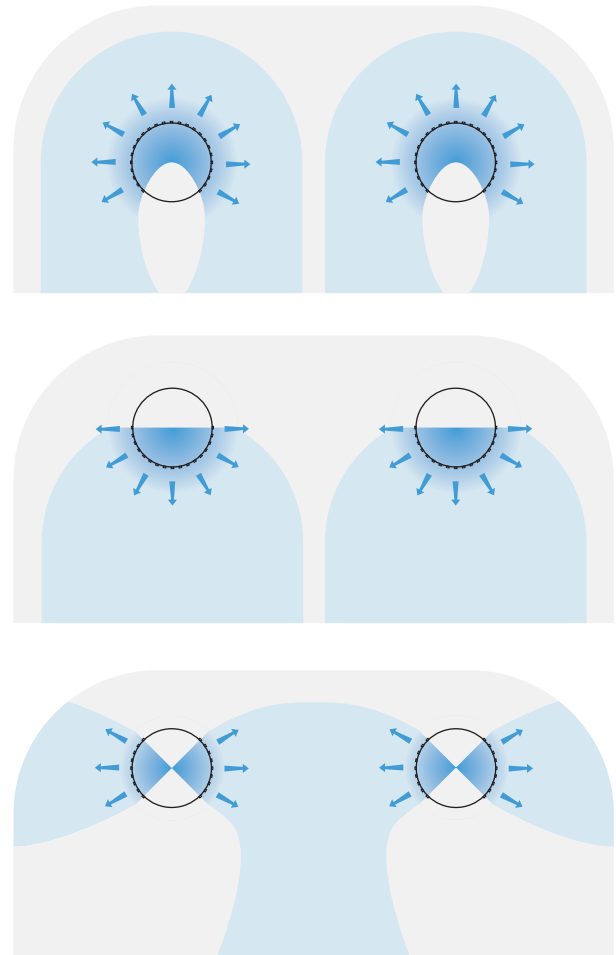
Downward supply air

When air is supplied downwards, the air velocities in the occupied zone are increased by the thermal forces (by cooling) and by the dynamic forces (Supplied air velocity). This can result in quite high air velocities in the occupied zone, which is not acceptable for traditional comfort ventilation. However, high air velocities can be recommended if a stable downward flow of air is required, and if increased, air velocities in the occupied zone are acceptable. This could, for example, be desirable for industrial assignments. A nozzle pattern between 90° and 300° is used, depending on the volume flow required.

Horizontal supply air

When air is supplied horizontally, air jets are formed, creating a mixed flow in the room. Depending on the various parameters, maximum air velocities occur in the occupied zone due to the thermal load, air jet velocities or a combination of both. When low supply air velocities are being used (low volume flow or large ducts/nozzle patterns) the form of the flow approximates a form of low impulse supply air, as with upwards supply air. Horizontal supply air can be used in locations where there is a deliberate demand for a flow of air throughout the room in accordance with the mixing principle, and therefore where an upward supply is not being used.

Dispersal patterns



Recommended working areas for Ventiduct

The values stated are for guidance only and should be used with care, as incoming volume flow, cooling temperature, duct design and air pattern all have a great deal of influence on the resulting air velocity in the occupied zone. For more detailed calculations, Lindab will be happy to carry out a computer calculation based on an actual installation.

Air pattern	Up	Down	Horizontal
Installation height [m] *	2,5-5,0	3,0-8,0	2,5-5,0
Min. distance from ceiling [m] **	0,2	0,1-0,2	0,1
$\Delta t (t_1 - t_r)$ [K]	-1..-10	-1..-6	-1..-8

* Distance from floor to lower edge of duct

** Distance from upper edge of duct to ceiling must be maintained to avoid dirtying the ceiling

Technical data

Max. volume flow per metre of duct (m³/h)

Dim	Nozzle pattern			
	90°	180°/2x90°	270°	300°
200	45	95	140	155
250	60	115	175	195
315	75	150	220	245
400	95	190	280	315
500	115	235	350	390

Max. total duct length (m)

Dim	Nozzle pattern			
	90°	180°/2x90°	270°	300°
200	14	7	5	4
250	17	8	6	5
315	21	11	7	6
400	27	14	9	8
500	34	17	11	10

Sound effect level L_w (dB) = $L_{WA} + K_{Ok}$

Dim	125	250	500	1K	2K	4K	8K
200	-7	0	1	-6	-15	-21	-27
250	-5	1	-1	-5	-11	-18	-22
315	1	2	-2	-4	-11	-16	-19
400	-1	-1	-3	-4	-9	-14	-17
500	4	0	-3	-4	-9	-16	-14

Technical data

Air velocity in the occupied zone

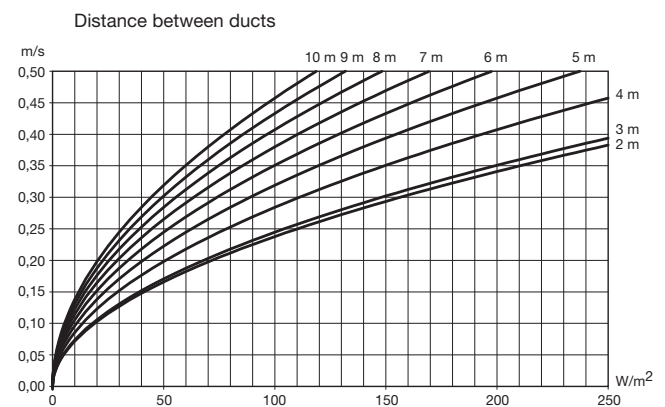
The air velocity in the occupied zone is a result of air jet velocities and thermal air movements in the room. An exact calculation of the resulting air velocity in the occupied zone can be performed using a computer program. (Contact the lindab sales department for further information).

For upward supply, the maximum air velocity in the occupied zone are dependent on the temperature difference $t_i - t_r$. The best results are achieved by using maximum supply air per duct metre, according to the table on the left.

Depending on the thermal load (W/m^2) and the duct length, the maximum air velocity in the occupied zone is indicated as a rough estimate in the diagram below.

Diagram only applies to upward dispersal pattern with maximum volume flow per duct metre:

(distance to ceiling > $4 \times \varnothing d$).

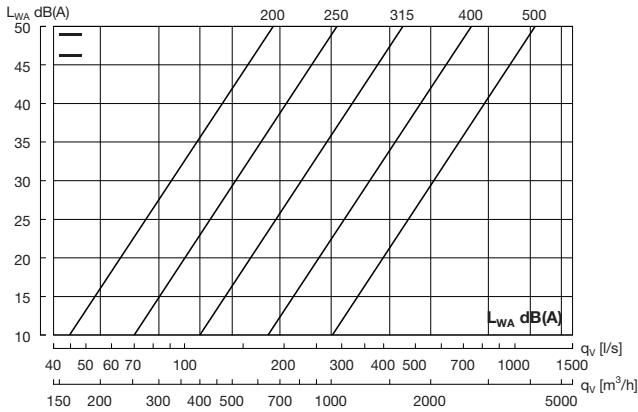


Please contact Lindab's sales department for further information.

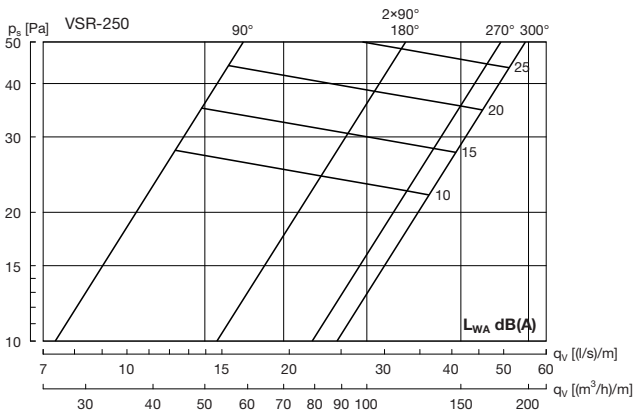
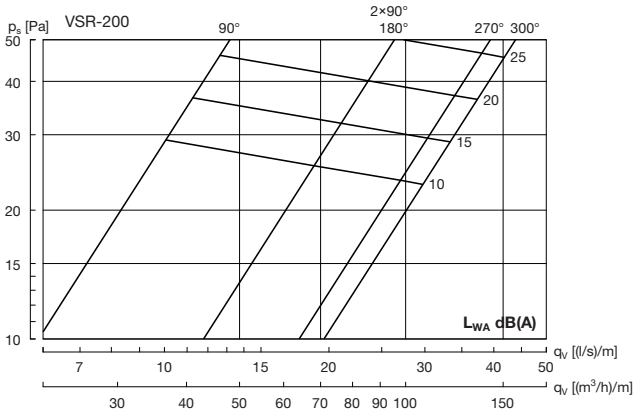
Pressure and sound

For calculation of the resulting sound power level from a ventiduct, add the sound power level from the nozzles (L_{WA} nozzles) and the sound power level from the flow noise in the ventiduct (L_{WA} duct) logarithmically.

Flow noise in duct



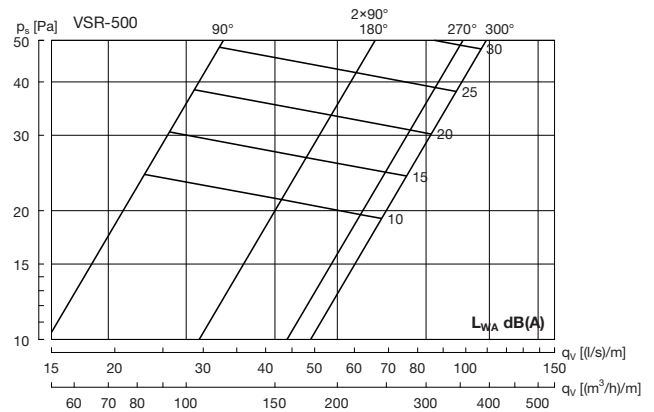
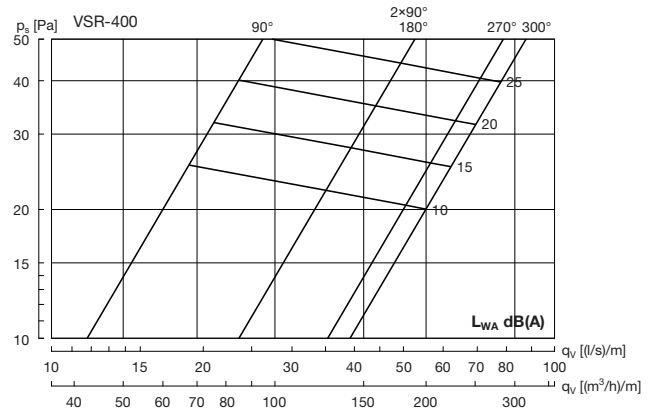
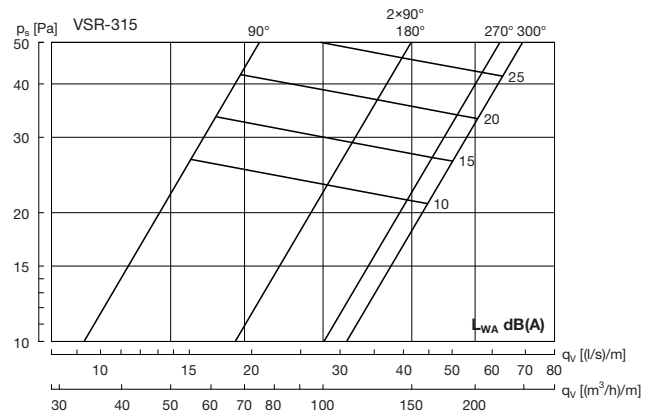
Sound effect level from nozzles



The sound levels from the nozzles apply for duct length 1 m

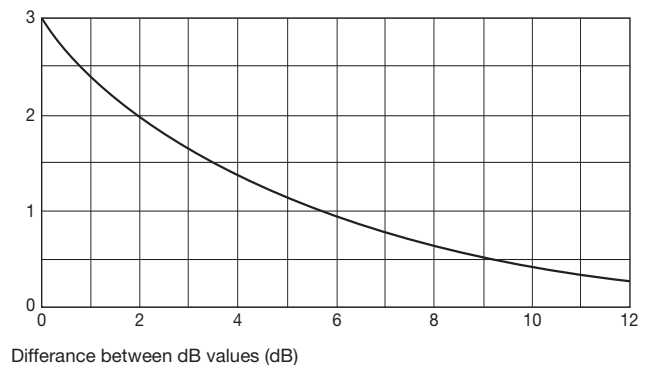
Correction for other duct lengths:

Length m	1,0	1,5	2,0	2,5	3,0	4,0	5,0	6,0
Corrections	0	2	3	4	5	6	7	8



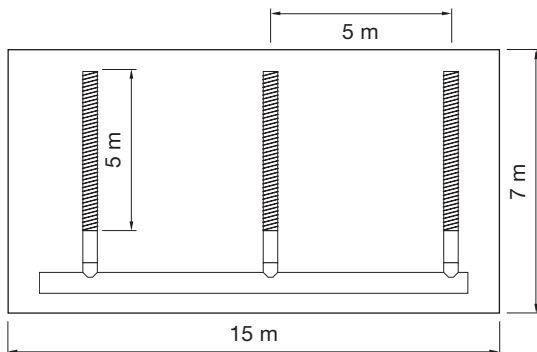
Addition of sound levels from nozzles and duct:

Difference added to highest dB value (dB)



Technical data

Calculation example



Required information:

Pressure loss: p_l [Pa]
 Resulting sound level in the rooms: L_p [dB(A)]
 Max. velocity in the occupied zone: v_{occ} [m/s]

Calculation based on catalogue values:

VSR-250, 270°
 Ceiling height: 5,0 m
 Installation height upper edge duct: 4,5 m
 Volume of the room: 525 m³
 Hard room: ($T_s \sim 1,9$ s)
 Volume flow: 2400 m³/h (667 l/s)

The following can be determined from the diagrams on the previous page:

Pressure loss: 40 Pa
 Sound effect: $L_{WA \text{ duct}}$: 41 dB(A)
 Sound effect: $L_{WA \text{ nozzle}}$: 22 dB(A)

Duct length of 5 m => correction of + 7

Sound effect nozzles corrected: $L_{WA \text{ nozzles}} = 22 + 7 = 29$ dB(A)

Addition of sound levels from nozzles and duct:

Difference: 12 dB -> No addition
 Three identical sound sources: + 4,8
 (see figure 25 in the Theory section)
 Sound effect L_{WA} for three ducts: 41 + 5 = 46 dB(A)

Resulting sound level:
 The sound formula from page 46 in the Theory section is used.

The absorption area of the room is determined by :

$$A = 0,16 (V/T_s) = 0,16 (525/1,9) = 44 \text{ m}^2 \text{ Sabine}$$

Based on Figures 27 and 28 in the Theory section, room attenuation D is determined:

Figure 27: $\sqrt{n}/\sqrt{Q} = 1,7$ for direction factor Q = 1 and n = 3
 1.5 m above the floor is distance to duct : $r = 4,5 - 0,25 - 1,5 = 2,75$ m

Figure 28: $r\sqrt{(n/Q)} = 4.7$ and $A = 44 \Rightarrow D = 10$ dB

Resulting sound pressure in the room:

$$L_p = L_{WA} \text{ (for three ducts)} - D = 46 - 10 = 36 \text{ dB(A)}$$

$$\Phi = 3,2 \text{ kW} \Rightarrow \Delta T = 3200 / (667 \cdot 1,2) = -4 \text{ K}$$

$$3200 \text{ W} / (15 \text{ m} \times 5 \text{ m})$$

=> 43 W/m² in the actively ventilated area

Speed in the occupied zone according to the diagram:

$$43 \text{ W/m}^2 \text{ and } 5 \text{ m distance} \Rightarrow v_{occ} = 0.21 \text{ m/s}$$

Dimensioning of Ventiduct

Projekt :

Produktionslokale				A B C					
Rum				Aktiv rumareal					
Længde	m	7	$A_{\text{total}} \text{ m}^2$	105,0	75	75	75		
Bredde	m	15	$V \text{ m}^3$	525	(areal)	ok	ok	ok	
Højde	m	3,5	Fri afstand		(bredde)	ok	ok	ok	
Opholdszone	m o. floor	1,5	ok rærløft		(længde)	ok	ok	ok	
Montagehøjde (ok) rør	m	4,5	ok	0,50	Maks. flow pr. m Ventiduct	m ³ /hm	175	175	175
Efterklangstid T_s	s	1,9	Rundtempring		Aktuelt flow pr. m rør	m ³ /hm	160	160	160
Absorptionskoefficient	α_a	0,10	hårdt		Kontrol maks. flow pr. m		ok	ok	ok
				Total retfængde					
				m 15,0 15,0 15,0					
				Kontrol maks. r/længde					
				ok ok ok ok					
				Alstand gulv / uk rør					
				m 4,25 4,25 4,25					
				Termiske parametre					
Dimension				Køling total					
Mønster				W 1632 3264 4896					
				Q/A W/m^2 16 31 47					
Spredning				Luftskifte					
				1/h 4,6 4,6 4,6					
				Flow pr A_{aktiv} m ³ /hm ² 32 32 32					
				Q/L W/m 109 218 326					
				Q/A _{aktiv} W/m^2 22 44 65					
Volumenstrøm (total)				Lyd					
m ³ /h 2400 2400 2400				Flow pr. rør m ³ /h 800 800 800					
Undertemperatur				Maks. kanalhastighed m/s 4,5 4,5 4,5					
K 2 4 6				Maks. kanal diameter dB(A) 30 30 30					
Antal rør stk. 3 3 3				kanal dB(A) 41 41 41					
Længde pr. rør m 5 5 5				Lydeffekt pr. rør dB(A) 42 42 42					
Afstand mellem rør eller Bln m 5 5 5									
Resultat									
Maks. hastighed m/s 0,15 0,21 0,25				Totaltrykfald Pa 53 53 53					
Resultierende lydtrykniveau dB(A) 36 36 36									
Bemærkninger									
				Indblæsning af 2400 m ³ /h i lokale på 15 x 7 m (LH = 5,0m)					
				Termisk belastning 3,2 KW					
				*Hårdt rum (efterklangstid: T_s 1,9 sek.)					

(Printout from the program)

Lindab is able to offer complete calculations for an actual installation using our internal dimensioning program (see printout above from the program). Based on the specification of a large number of variables, detailed information can be obtained on maximum velocities in the occupied zone, pressure loss and resulting sound levels in the rooms for the overall installation. Variables that it is not possible to include in calculations based on the catalogue values.

Contact Lindab for further information.

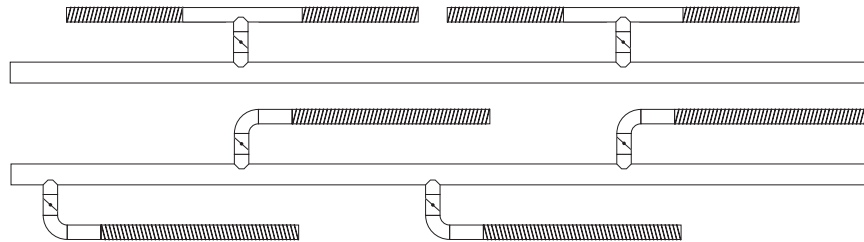
Technical data

Examples of duct design

Ventiduct nozzle ducts can be installed in various ways. In high-ceilinged rooms it is generally an advantage to install Ventiduct nozzle ducts as low down as possible (min. height above floor 2.5 m). This provides the greatest efficiency.

Cactus model

This solution is used for long, narrow rooms.



Exchange model

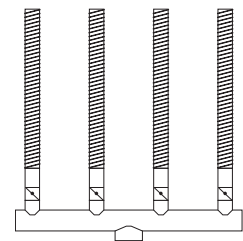
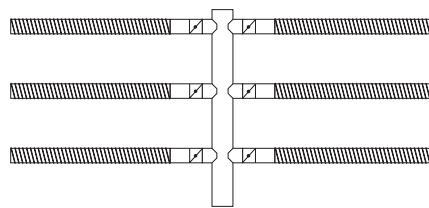
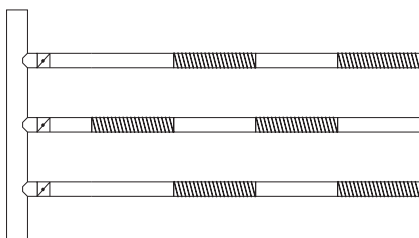
An ideal solution for long, narrow rooms. This model provides an even distribution of supplied air.

Fishbone model

Ventiduct nozzle ducts stretch out from both sides of the main duct. It is recommended that an adjustment damper be used for accurate regulation of the air volume.

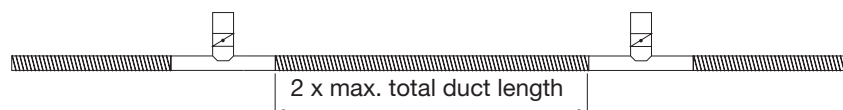
Fork model

Here the Ventiduct nozzle ducts are positioned on one side of a main or branch duct. It is recommended that an adjustment damper be installed on the duct joints in order to ensure consistent air distribution in the duct system.

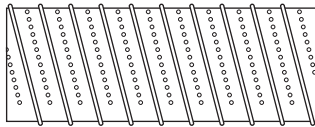


Line model

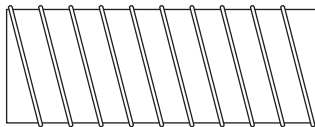
A simple solution that makes duct installation easier and minimises the number of adjustment dampers. The distance between the connection ducts is equivalent to twice Ventiduct's maximum length plus the two blind pieces.



Components



VSR nozzle duct - Nozzle pattern 90 - 300
Ventiduct nozzle ducts over 3 m are supplied in multiple sections, e.g. one 4 m long duct is supplied in two 2 m lengths.



VSR 000
Blind piece without nozzles, spiral-seamed.

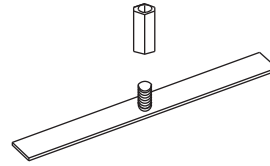


VSR 001
Blind piece without nozzles, long-seamed (smooth)

Order code

Product	INV	aaa
Type		
Dimension Ød		

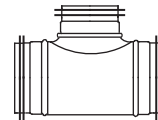
Accessories



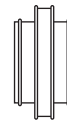
INV
Mounting bracket for Ventiduct



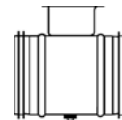
OSB10
Threaded rod



TCPU
T-piece



DIRU
Iris damper



DRU
Balancing damper



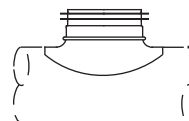
NPU
Spigot



ESU
End cap



ESUH
End cap with handle



PSU
Saddle

All accessories are supplied in the same material as the Ventiducts, and can also be supplied with a powder-coated finish.

Other components

Motorised shut-off and adjustment damper DCT and volume flow regulator VRU incl. accompanying silencer SLU.

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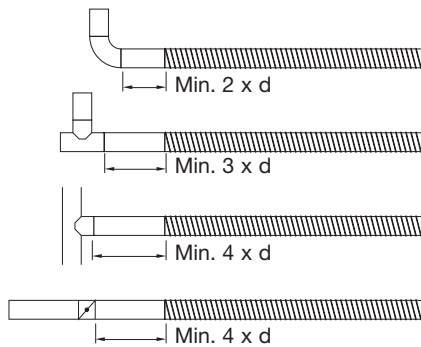
18

Technical data

Building-in distance

Ventiducts should not be positioned too close to dampers, bends, T-pieces or other elements that may create turbulence and hence noise.

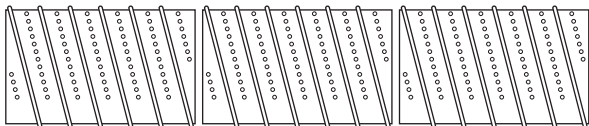
Straight duct sections should be installed between the Ventiducts and potentially disruptive components, as shown in the illustration below. Suitable duct sections are available.



Mounting

Assembly

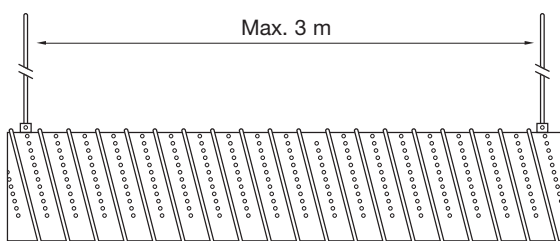
The Ventiducts are individually packed in cardboard boxes at the factory, to minimise the risk of transport damage. The packaging is numbered to ensure that the ducts are mounted in the correct order, so that the spiral seam is continuous.



Suspension

If it is necessary to be able to dismantle the Ventiducts, e.g. for cleaning, we recommend using Lindab Transfer connections (see Lindab's Duct Systems catalogue)

IMPORTANT: In order to maintain the number sequence, the Ventiducts should be left in their packaging until mounting commences.



Maximum distance between suspension loops is 3 metres.

Balancing

Measuring of the airflow

The easiest way to measure the volume flow is to measure the nozzle pressure in the middle of the Ventiduct (see sketch).

To do this, attach the hose from the manometer to one of the nozzles. The static pressure (P_s) in the duct can then be read.

Once you know the static pressure, you can read the volume flow per m/duct from the "Sound and pressure" diagram for the relevant duct dimension and nozzle pattern.

The total volume flow can thus be calculated by multiplying the relevant diagram value by the total active length of the Ventiduct.

